

A motor vehicle wheel hub bearing and a method for mounting the bearing onto a motor vehicle suspension

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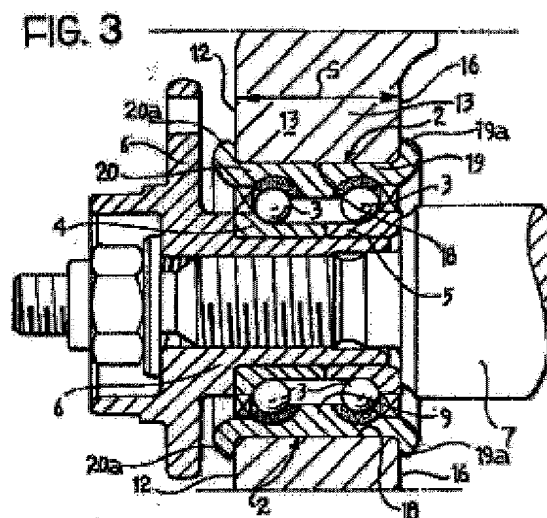
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EP0189365
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Abstract of EP1031439

A motor vehicle wheel hub bearing comprises an outer race (2) which can be mounted in a cylindrical seat (14) formed in a suspension strut (13) and fixed in the seat by cold rolling. The outer race has an outer cylindrical surface (18) adapted to be fitted into the seat (14). The cylindrical surface (18) extends over the whole axial dimension of the outer race (2) and has a greater axial dimension than that of the strut (13). The two axially opposite ends of the cylindrical surface (18) are constituted by respective axial tubular projections (19), (20) adapted to be deformed by cold forming radially outwardly towards respective opposite lateral surfaces (16, 12) of the strut (13) so as to fix the bearing to the strut.



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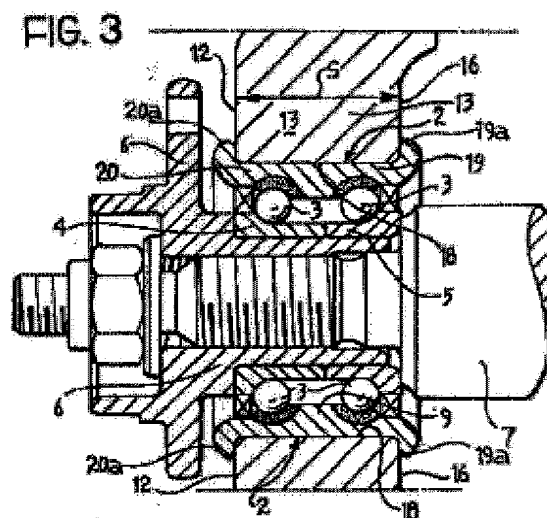
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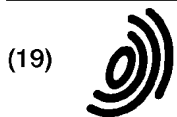
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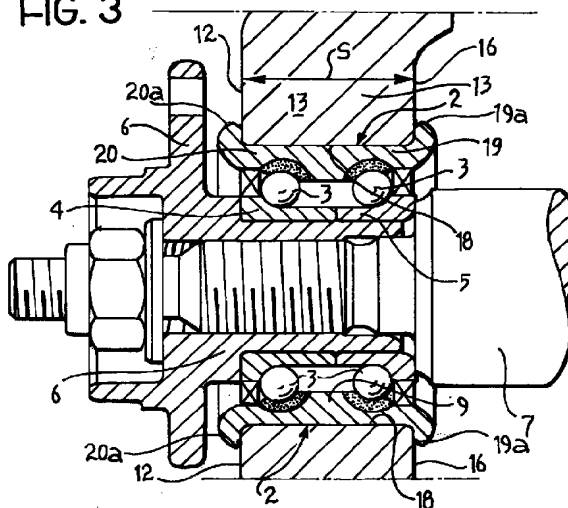
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(54) **A motor vehicle wheel hub bearing and a method for mounting the bearing onto a motor vehicle suspension**

(57) A motor vehicle wheel hub bearing comprises an outer race (2) which can be mounted in a cylindrical seat (14) formed in a suspension strut (13) and fixed in the seat by cold rolling. The outer race has an outer cylindrical surface (18) adapted to be fitted into the seat (14). The cylindrical surface (18) extends over the whole axial dimension of the outer race (2) and has a greater axial dimension than that of the strut (13). The two axially opposite ends of the cylindrical surface (18) are constituted by respective axial tubular projections (19), (20) adapted to be deformed by cold forming radially outwardly towards respective opposite lateral surfaces (16, 12) of the strut (13) so as to fix the bearing to the strut.

FIG. 3



Description

[0001] The present invention relates to a motor vehicle wheel hub bearing of the type identified in the preamble of Claim 1. The invention further relates to a method for mounting the bearing on the suspension strut of a motor vehicle.

[0002] A bearing of the said type, known from European Patent Application EP-A-0 794 072 is illustrated in Figure 1. With reference to Figure 1, a bearing generally indicated 1 comprises a radially outer race 2 which has two rolling raceways for respective sets of rolling balls 3. The balls can rotate on radially inner raceways 4 and 5 which are mounted on a flanged hub 6 fixed for rotation in a known way to an axle 7.

[0003] The outer race 2 includes an annular main portion 6 which extends to only one axially inner side (that is to say facing towards the interior of the vehicle) forming an axial tubular projection 10 which defines, with the main annular portion 9, an axial abutment shoulder 11 able to abut against the outer lateral surface 12 of the strut 13.

[0004] A cylindrical seat 14 is formed in the strut 13, into which the outer cylindrical surface 15 of the tubular projection 10 is force fitted. This tubular projection has an axial dimension greater than the axial thickness of the strut 13, in such a way that the axially inner rim of the tubular projection 10 can be folded and radially outwardly deformed against the axially inner lateral surface 16 of the strut 13 by means of a cold-rolling plastic deformation operation, to obtain a rolled edge 17 which thus axially fixes the bearing onto the suspension strut.

[0005] This arrangement, which is advantageous as it makes it possible to fix the bearing onto the suspension without causing deformations in the region of the raceways of the outer race of the bearing, does, however, have a disadvantage due to the fact that it requires very accurate mechanical finishing in particular for the formation of the shoulder 11. The outer race of the bearing is moreover made heavier by that portion of the material which forms the said shoulder.

[0006] An object of the invention is to provide a bearing of the above-identified type, having an outer race of less weight and which involves, for its finishing, a simpler and more rugged turning operation, which is therefore economically advantageous.

[0007] Another object of the invention is to provide a bearing in which the material from which it is made is utilised in an optimal manner.

[0008] A yet further object is to centre a bearing in an improved and more "flexible" manner on the motor vehicle suspension.

[0009] These objects are achieved according to the present invention by a bearing having the characteristics set out in Claim 1.

[0010] According to another aspect of the invention, these objects are achieved by a method as defined in Claim 6.

[0011] Further important characteristics of the invention are specified in the dependent claims.

[0012] The characteristics and advantages of the present invention will become apparent from the detailed description of various embodiments thereof, given with reference to the attached drawings, given by way of non-limitative example, in which:

Figure 1 is a vertical axial section of a prior art wheel hub bearing mounted on a motor vehicle strut;

Figure 2 is a partial axial section of a detail of the bearing of Figure 1;

Figure 3 is an axial sectional view of a motor vehicle wheel hub having a bearing according to the present invention fixed to the suspension strut of a motor vehicle;

Figure 4 is a view in axial section of the outer race of the bearing of Figure 3;

Figure 5 illustrates a mounting phase of the bearing onto the suspension; and

Figure 6 is a partial view in axial section of a forged blank from which an outer race for a bearing according to the invention is formed.

[0013] Referring to Figures 3 and 4, and utilising the same reference numerals already adopted for Figure 1, a bearing according to the present invention comprises a radially outer race 2, described in more detail below, a pair of radially inner races or half-races 4 to 5, and two sets of rolling elements 3, in this example balls, radially interposed between the outer race 2 and the inner half-races 4 and 5.

[0014] Making reference now in particular to Figure 4, the outer race 2 comprises a central main annular portion 9 in which the raceways are formed for the rolling elements 3. The outer race 2 has an outer cylindrical surface 18 adapted to be inserted in a cylindrical seat 14 formed in the strut 13 of a suspension. Preferably, the seat 14 and the cylindrical surface 18 are so dimensioned as to provide some radial interference requiring a forced fitting of the bearing in the strut, to obtain a better locking of the bearing.

[0015] The outer surface 18 extends over the whole of the axial dimension of the outer race 2 and has a greater axial dimension than that of the strut 13.

[0016] The two axially opposite ends of the cylindrical surface 18 comprise respective axial tubular opposite end projections 19 and 20 which extend from the central main portion 9; the tubular projections 19 and 20 are shaped and outwardly dimensioned to be plastically deformed, cold, preferably by being rolled radially outwardly towards the inner and outer lateral surfaces 16 and 12 of the strut 13 so as to assume the configuration illustrated in Figure 3, and in broken outline in Figure 4, as indicated by the reference numerals 19a and 20a.

[0017] Preferably, the radially inner races 4 and 5 have an overall axial dimension not greater than the

axial thickness s of the strut 13.

[0018] The sequence of operations for mounting the bearing onto the strut provides that the complete bearing of outer race, inner race and rolling elements is inserted into the seat 14 of the strut. In the preferred embodiment, in which there is provided some radial interference between the surface 18 of the outer race 2 and the seat 14 of the strut, the outer race 2 is inserted forcibly. The insertion could for example be effected by means of a tool 21 schematically illustrated in Figure 5 which has a first axial abutment surface 22 and a second axial abutment surface 23, axially spaced from one another by a predetermined distance e .

[0019] The first abutment surface 22 presses the outer, undeformed face 24 of the axially outer extension 20 in the axial direction indicated by the arrow A until the second axial abutment surface 23 engages against a reference surface 25 fixed to the strut, in this case represented by a zone 12 of the outer surface of the strut close to the cylindrical seat 14.

[0020] The distance e is predetermined in such a way that in the abutment configuration the ends of the tubular projections 19 and 20 project, in the undeformed condition before rolling, by respective distances e' and e'' beyond the lateral surfaces 16 and 12 of the strut. The distances e' and e'' can be the same or slightly different depending on requirements, thus making it possible to fix the bearing to the strut in a plurality of different axial positions. In each case the distances by which the said undeformed ends project beyond the lateral surfaces of the strut will be such as to allow the formation of corresponding rolled edges 19a and 20a effective in axially fixing of the bearing onto the strut.

[0021] The rolling operations can be equally well performed simultaneously on both projections or, alternatively, in two successive steps, resisting axial movement of the outer race 2 by means of the same tool 21 utilised to determine the desired axial position of the outer race with respect to the strut.

[0022] In the preferred embodiment of the method according to the invention, the deformed edges 19a and 20a each have an axial thickness b less than or equal to the radial thickness t of the tubular projections 19 and 20 in their undeformed condition, as indicated in Figure 4.

[0023] For the purposes of improving the fixing of the bearing to the strut it is also advantageous to form a bend or a circumferentially curved connection 26 on the strut, in the connection region between the opposite lateral surfaces 16, 12 and the cylindrical seat 14, each preferably having an axial extent h not greater than 3 mm.

[0024] As may be appreciated, the outer race of the bearing according to the present invention is obtained from a forged blank of very much simpler form, as schematically indicated by the broken outline C in Figure 6. It will be noted that on the radially outer surface of the outer race it is not necessary to form with precision the

conventional shoulder 11 of Figure 2.

[0025] Moreover, with particular reference to Figure 2, the material conventionally used to form the reinforcement region 11a of the shoulder 11 is saved: as a consequence, the working operations will be expedited with respect to the prior art discussed in the introduction, in that the turning operation to obtain a cylindrical surface is simpler and faster.

[0026] In addition, the quantity of excess metal to remove from the partly worked workpiece will generally be less as will be apparent from comparison of the broken outlines in Figures 2 and 6.

[0027] Finally, the outer races obtained according to the invention are lighter and have an axially symmetrical structure so that they can be mounted indifferently with either of their faces being the frontal face thus simplifying mounting.

[0028] Naturally, the principle of the invention remaining the same, its details can be varied with respect to what has been described in the present example. In particular, it would be possible to form a plurality of radial recesses (or projections) in one or both lateral surfaces 12 and 16 of the strut, which would be interpenetrated by the folded material of one or both edges 19a or 20a so as to ensure fixing against possible rotation of the outer race with respect to the strut.

Claims

1. A bearing for a motor vehicle wheel hub, comprising an outer race (2) which can be mounted in a cylindrical seat (14) formed in a suspension strut (13) and fixed in the said seat by cold forming, in which the said outer race (2) has:

a main annular portion (9) forming raceways for rolling elements (3);
an outer cylindrical surface (18) to be fitted in the said seat (14);
characterised in that the cylindrical surface (18) extends over the whole axial dimension of the outer race (2) and has an axial dimension greater than that of the strut (13), and in that the two opposite axial ends of the cylindrical surface (18) comprise respective axial tubular projections (19, 20) which extend from the said main portion (9); the said tubular projections (19, 20) being dimensioned so that their ends can be plastically deformed by cold forming radially outwardly towards respective opposite lateral surfaces (16, 12) of the strut so as to fix the bearing to the strut.

2. A bearing according to Claim 1, characterised in that the said main annular portion (9) is substantially central.
3. A bearing according to Claim 2, characterised in

that the said outer race (2) has an axially symmetrical shape.

4. A bearing according to Claim 1, characterised in that the said tubular projections (19, 20) are dimensioned in such a way that in the undeformed condition before being cold formed their opposite ends extend beyond the said lateral surfaces (12, 16) by respective distances (e', e'') so as to allow the bearing to be fixed to the strut in a plurality of axially different positions. 10
5. A bearing according to Claim 1, characterised in that it comprises one or more radially inner races (4, 5) having an overall axial dimension not greater than the axial thickness (s) of the strut (13). 15
6. A method for mounting a motor vehicle wheel hub bearing to a suspension strut, the method comprising the following steps: 20
 - (a) providing a strut (13) of a suspension having an axial cylindrical seat (14) extending between two opposite lateral surfaces (16, 12);
(b) providing a bearing comprising an outer race (2) having:
a main annular portion (9) forming raceways for rolling elements (3); and
an outer cylindrical surface (18) which can be inserted into the said seat (14) of the strut, the said cylindrical surface (18) extending over the whole of the axial dimension of the outer race (2) and having a greater axial dimension than the axial thickness of the strut (13); the two opposite axial ends of the cylindrical surface (18) being constituted by respective axial tubular projections (19, 20) which extend from the said main portion (9); 30
(c) inserting the outer ring (2) into the seat (14) of the strut in such a way as to leave opposite end portions of the tubular projections (19, 20) projecting beyond the said opposite lateral surfaces (16, 12) by predetermined distances (e', e''); 35
(d) cold forming the said end portions in radially outwardly directions towards the said respective opposite lateral surfaces (16, 12) of the strut (13), thereby forming deformed edges (19a, 20a) which fix the bearing to the strut (13). 40
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7. A method according to Claim 6, characterised in that the said cold forming step (d) forms deformed edges (19a, 20a) each having an axial thickness (b) less than or equal to the radial thickness (t) of the 55

tubular projections (19, 20) in the undeformed condition.

8. A method according to Claim 6, characterised in that the said step (a) further comprises the step of:
(a1) forming chamfers, bevels or circumferentially curved connections (26) on the strut (13), in correspondence with the connection region between the said opposite lateral surfaces (16, 12) and the said cylindrical seat (14). 10
9. A method according to Claim 8, characterised in that the said chamfers, bevels or curved connections (26) each have an axial extent (h) not greater than 3 mm.
10. A method according to Claim 6, characterised in that the said cold deformation step (d) further comprises the following steps in succession:
(d1) deforming one of the tubular end portions (19) while resisting axial movement of the outer race (2) by means of a tool (21) acting on the other tubular end portion and shaped (22, 23) to guarantee the desired axial positioning of the outer race (2) with respect to the strut (13);
(d2) subsequently deforming the end of the other tubular portion (20). 15
11. A method according to Claim 6, characterised in that in the said cold deformation step (d) both the ends of the tubular portions (19, 20) are simultaneously deformed.
12. A method according to Claim 6, characterised in that the said insertion step (c) is effected by utilising a tool (21) which has
a first axial abutment surface (22) able to engage an axially outer side (24) of the axial outer tubular projection (20); and
a second axial abutment surface (23) able to engage against a reference surface (25) fixed to the strut, said first and second surfaces (22, 23) being axially spaced from one another by a predetermined distance (e) to position the opposite ends of the tubular projections (19, 20) to project by respective predetermined distances (e', e'') beyond the said lateral surfaces (16, 12) of the strut. 20
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13. A method according to Claim 6, characterised in that the said insertion step (c) is effected by utilising a tool (21) which has
a first axial abutment surface (22) able to engage an axially outer side (24) of the axially 60

outer tubular projection (20); and
a second axial abutment surface (23) able to
engage against a reference surface (25) fixed
to the strut, said first and second surfaces (22,
23) being axially spaced from one another by a
predetermined distance (e) to position the
opposite ends of the tubular projections (19,
20) to project by respective predetermined dis-
tances (e', e'') beyond the said lateral surfaces
(16, 12) of the strut.

14. A method according to Claim 6, 12 or 13, character-
ised in that the said distances (e', e'') are equal.

15. A method according to Claim 6, 12 or 13, character-
ised in that the said distances (e', e'') are different
from one another.

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FIG. 1 *PRIOR ART*

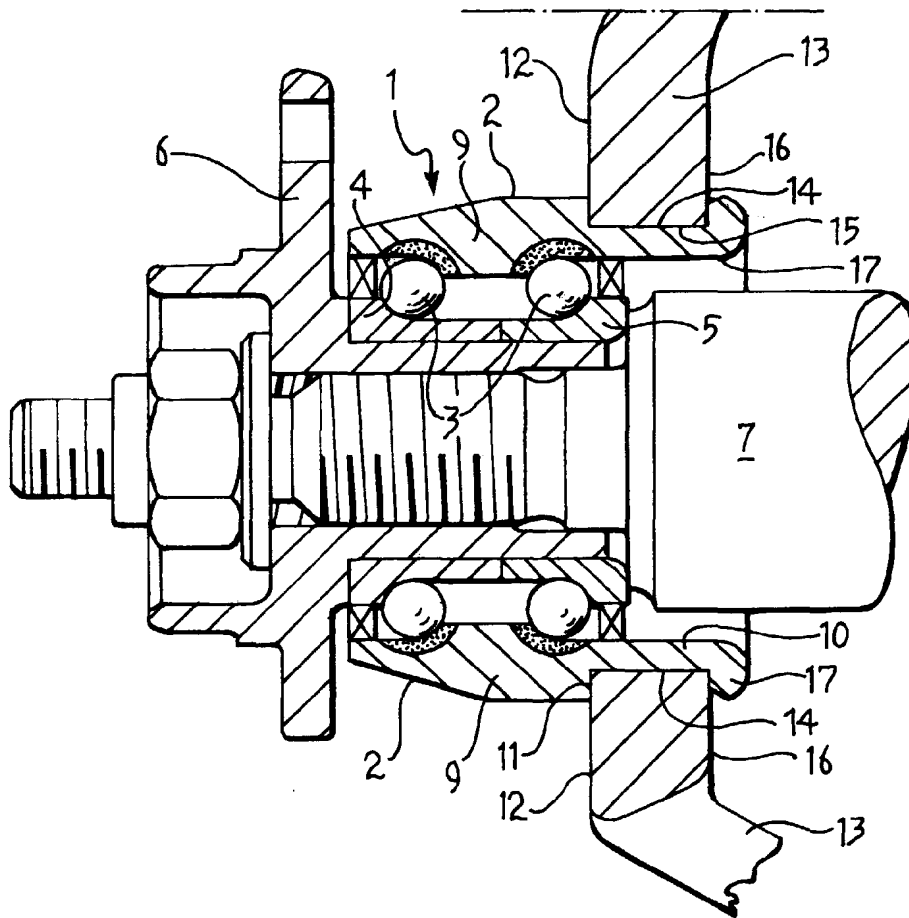


FIG. 2 *PRIOR ART*

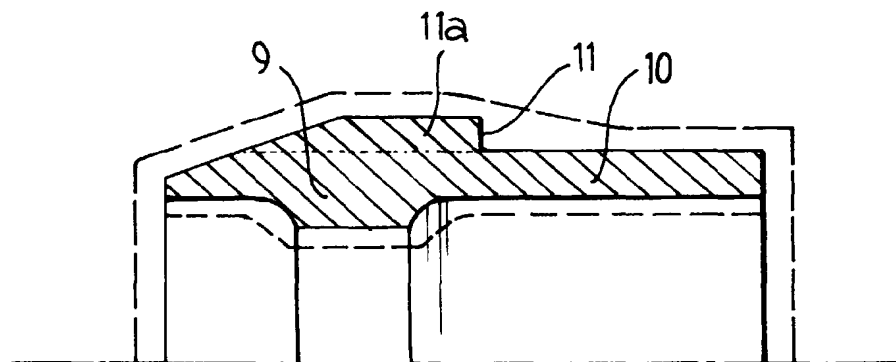


FIG. 3

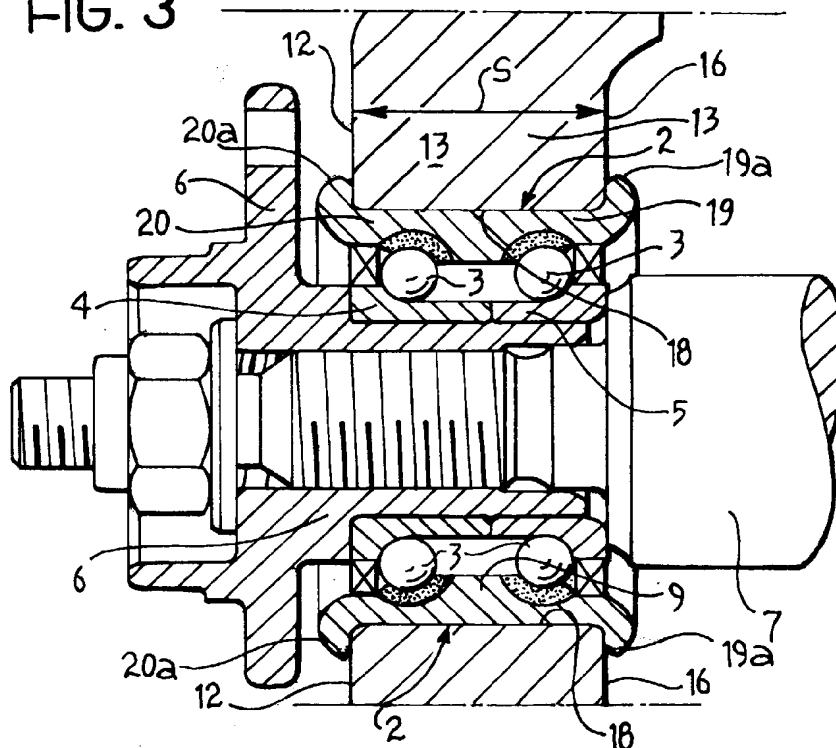


FIG. 4

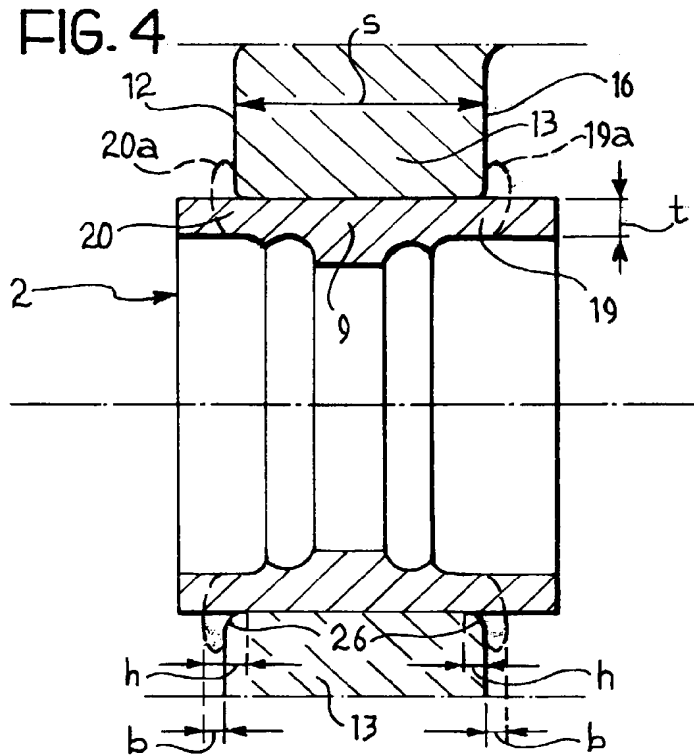


FIG. 5

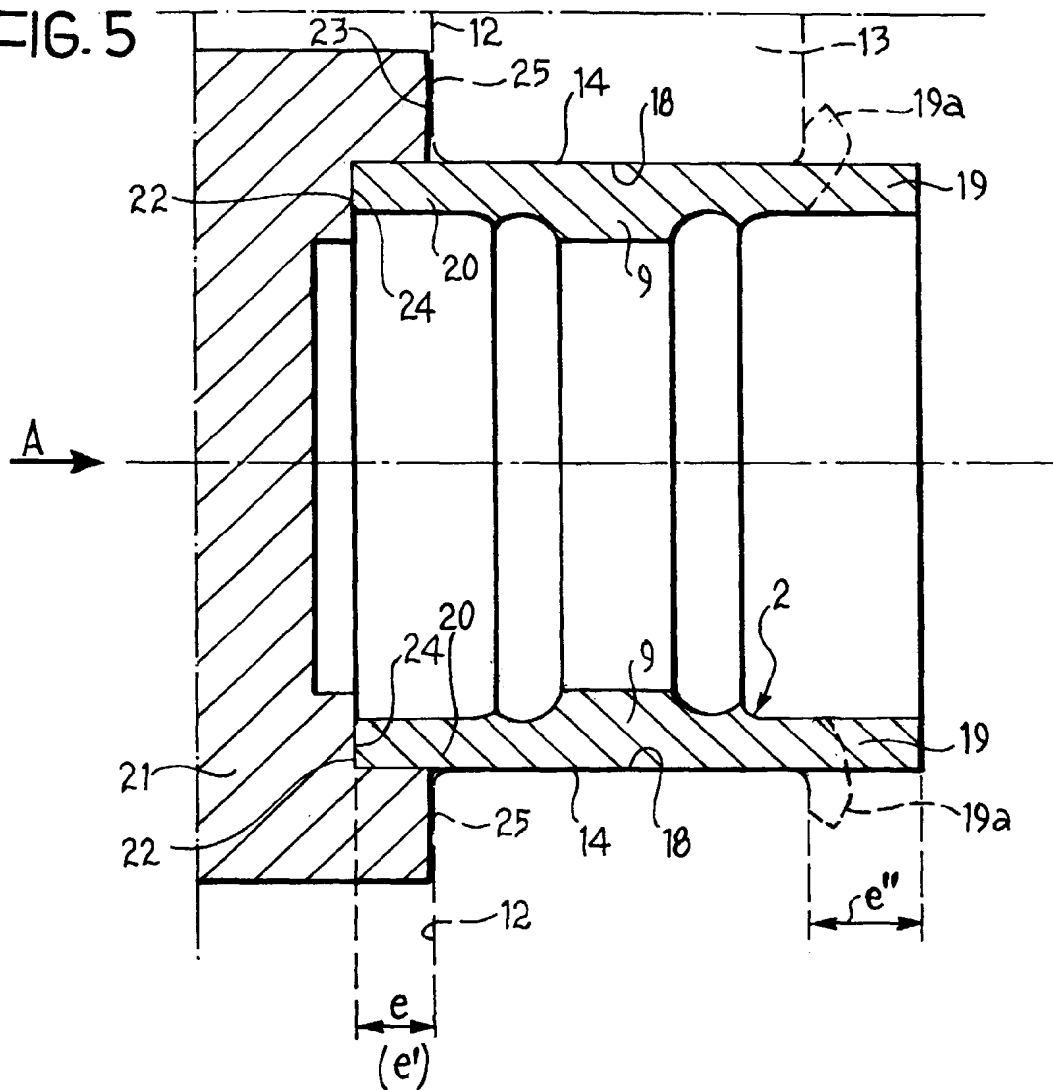


FIG. 6

